

系所別	組別	考試科目 (中文名稱)	考試日期	節次	備註
機械工程研究所	乙組	流體力學或熱力學	3月27日	第2節 10:30~12:00	可攜帶不可 程式之 計算本

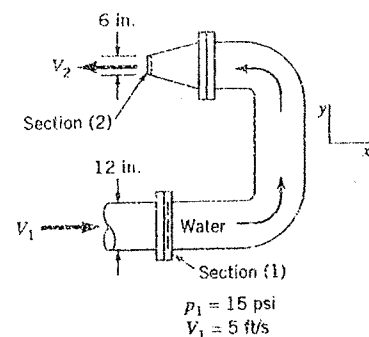
註：考生可否攜帶計算機或其他資料作答，請在備註欄註明（如未註明，一律不准攜帶）

請考生特別注意：請由以下題目中任選5題作答(答題數不可超過5題，第6題(含)以上不予計分)。
並請詳列計算步驟，否則一概不計分。

- (20%) Answer the following questions.
 - (5%) What is a "fluid"? Give word statement. Do not give any example.
 - (6%) Write the Navier-Stokes equation in vector form for Newtonian fluid with constant viscosity, assuming gravitational force is the only body force. Write also the x -component of the equation in Cartesian coordinates.
 - (9%) Give a schematic diagram of a Pitot tube, and derive from which a formula for the calculation of flow velocity based on the measured data.

- (20%) The velocity potential for a given irrotational, two-dimensional flow field is $\phi = \left(\frac{5}{3}\right)x^3 - 5xy^2$.
 - Determine the velocity components u and v .
 - Show that the differential form of the continuity equation is satisfied.
 - Determine the corresponding stream function.

- (20%) Determine the magnitude and direction of the x and y components of the anchoring force required to hold in place the horizontal 180° elbow and nozzle combination shown in the figure to the right. Neglect gravity and use $\rho = 1.94$ slugs/ft³ for water density.



- (20%) For the fluid flow in a smooth tube, the pressure drop per unit tube length (Δp_t) can be expressed in the following function form: $\Delta p_t = f(D, \rho, \mu, V)$, where D is the inner diameter of the tube, ρ is the fluid density, μ is the dynamic viscosity of the fluid, V is the averaged flow velocity. Use the MLT dimensional system in the following analysis. (a) List the MLT dimensions of the above five variables, (b) Write the MLT dimensional matrix for the above five variables and show that the rank of the dimensional matrix is 3, (c) Choose $D \cdot \rho \cdot V$ as the repeating variables and use the Buckingham Pi theorem to find appropriate dimensionless products (Pi terms).
- (20%) Consider the laminar flow of an incompressible fluid past a flat plate at $y = 0$. The boundary layer velocity profile is approximated as $u = Uy/\delta$ for $0 \leq y \leq \delta$ and $u = U$ for $y > \delta$. (a) Determine the momentum thickness. (b) Determine the wall shear stress by using the momentum integral equation.
- (20%) Nitrogen gas contained in a piston-cylinder device is compressed from 0.5 to 0.1 m³. During the process, the pressure and volume are related by $PV^2 = a$, where $a = 6$ kPa·m⁶. Please determine the work done on the nitrogen gas during this process?

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7. (20%) A piston-cylinder device contains 5 kg of refrigerant-134a at 1.2 MPa and 100 °C. Initially, the enthalpy (h) and internal energy (u) of the refrigerant are 332.47 and 305.54 kJ/kg, respectively. The refrigerant is now cooled at constant pressure until it exists as a saturated liquid-vapor mixture with a quality of 60%. Determine the amount of heat loss and show the process on a T- v diagram with respect to saturated lines.

Saturated refrigerant-134a: Pressure table

Pressure (MPa)	Internal Energy, kJ/kg		Enthalpy, kJ/kg	
	Sat. liquid, u_f	Sat. vapor, u_g	Sat. liquid, h_f	Sat. vapor, h_g
1.0	104.42	247.77	105.29	267.97
1.2	114.69	251.03	115.76	270.99
1.4	123.98	253.74	125.26	273.40

8. (20%) A heat engine operates between two reservoirs at 800 and 20 °C. One-half of the work output of the heat engine is used to drive a Carnot heat pump that removes heat from the cold surroundings at 2 °C and transfers it to a house maintained at 22 °C. If the house is losing heat at a rate of 62000 kJ/h, determine the minimum rate of heat supply to the heat engine required to keep the house at 22 °C.
9. (20%) Air enters the compressor of a gas-turbine plant at ambient conditions of 100 kPa and 27 °C with a low velocity and exists at 1 MPa and 347 °C with a velocity of 90 m/s. The compressor is cooled at a rate of 1500 kJ/min, and the power input to the compressor is 250 kW. Determine the mass flow rate of air through the compressor.

Table: Ideal-gas properties of air

Temperature (K)	Enthalpy (h), kJ/kg	Internal Energy (u), kJ/kg
300	300.19	214.07
320	320.29	228.42
600	607.02	434.78
620	628.07	450.09

10. (20%) Consider a steam power plant operating on the simple ideal Rankine cycle. The steam enters the turbine at 3 MPa and 350 °C and is condensed in the condenser at a pressure of 75 kPa. The enthalpy (h) and specific volume (v) of the water at the pump inlet are 384.39 kJ/kg and 0.001037 m³/kg, respectively. The enthalpy (h) and entropy (s) of the steam at the turbine inlet are 3115.3 kJ/kg and 6.7428 kJ/kg·K, respectively. Determine the thermal efficiency of this cycle.

Saturated water: Pressure table

Pressure (kPa)	Enthalpy, kJ/kg		Entropy, kJ/kg·K	
	Sat. liquid, h_f	Sat. vapor, h_g	Sat. liquid, s_f	Sat. vapor, s_g
50	340.49	2645.9	1.0910	7.5939
75	384.39	2663.0	1.2130	7.4564
100	417.46	2675.5	1.3026	7.3594